



Nonlinearity of Joints in Structural Dynamics of Weapons Systems

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AWE

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[†] Sandia National Laboratories is a multi-program laboratory managed and operated by Sandia Corporation, a wholly owned subsidiary of Lockheed Martin Corporation, for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-AC04-94AL85000.





WHY THIS IS IMPORTANT

- Joints are a (the) major source of variability and nonlinearity in our structures.
- Linear models are incorrect. Calibration in one experiment yields predictions that do not match other experiments.
- Propagation of parameter uncertainty with the wrong model form is nonsense.
- Tuning linear models to small-amplitude tests yields over-conservative models. Affordable designs are scrapped.
- Even though linear models are usually conservative - this is not always the case!



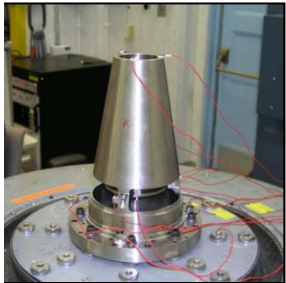
What we can do?

	Single Homogeneous Structure	Simple Assembly Level	Complicated Assembly Level
Natural Frequencies	✓	✓	✓
Mode Shapes	✓	✓	✓
Identify problem Frequencies	✓	✓	Depending on complexity
Amplitude	✓	✗	✗
Cumulative effects	Depending on problem	✗	✗

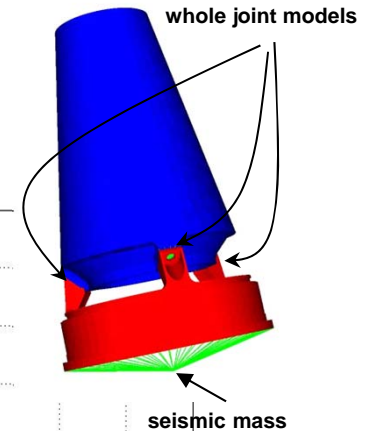
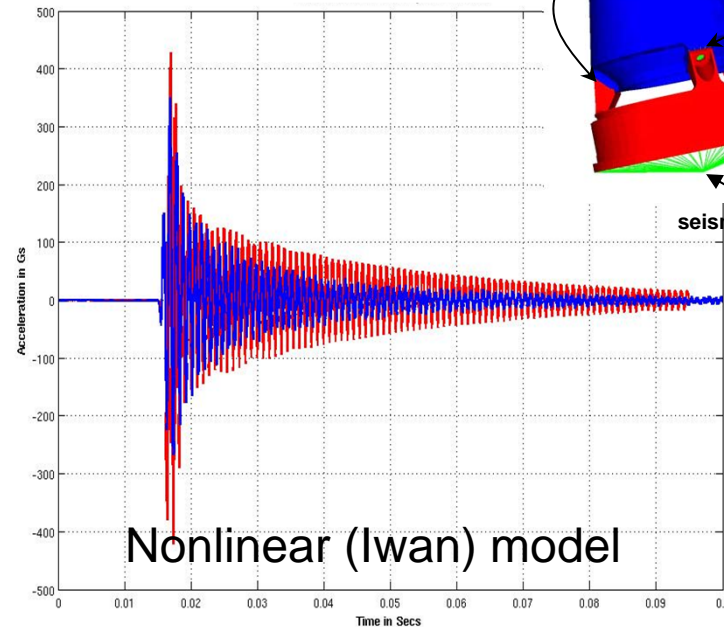
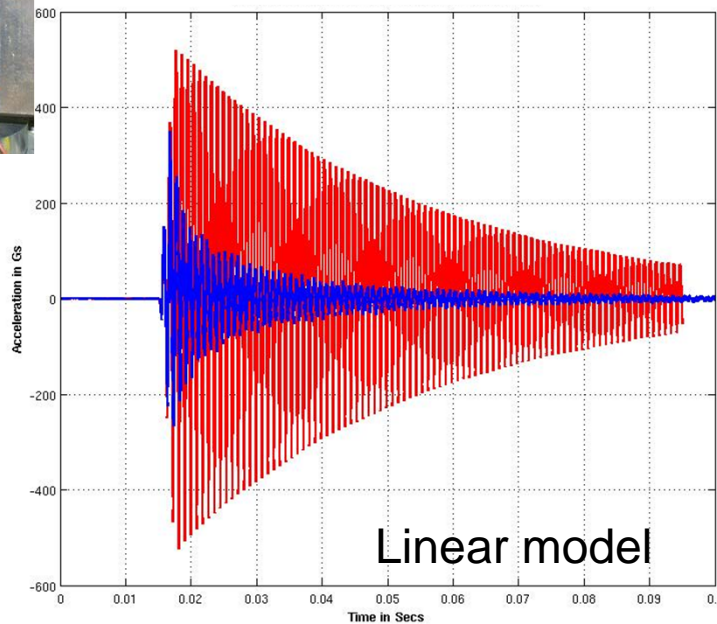


Even Simplest Systems are a Challenge

- Macro-slip and effective vibration isolation during blast
- High damping during sustained excitation



Acceleration predictions at forward mount joints:
Ti-SS mock 3-leg with shaker dynamics



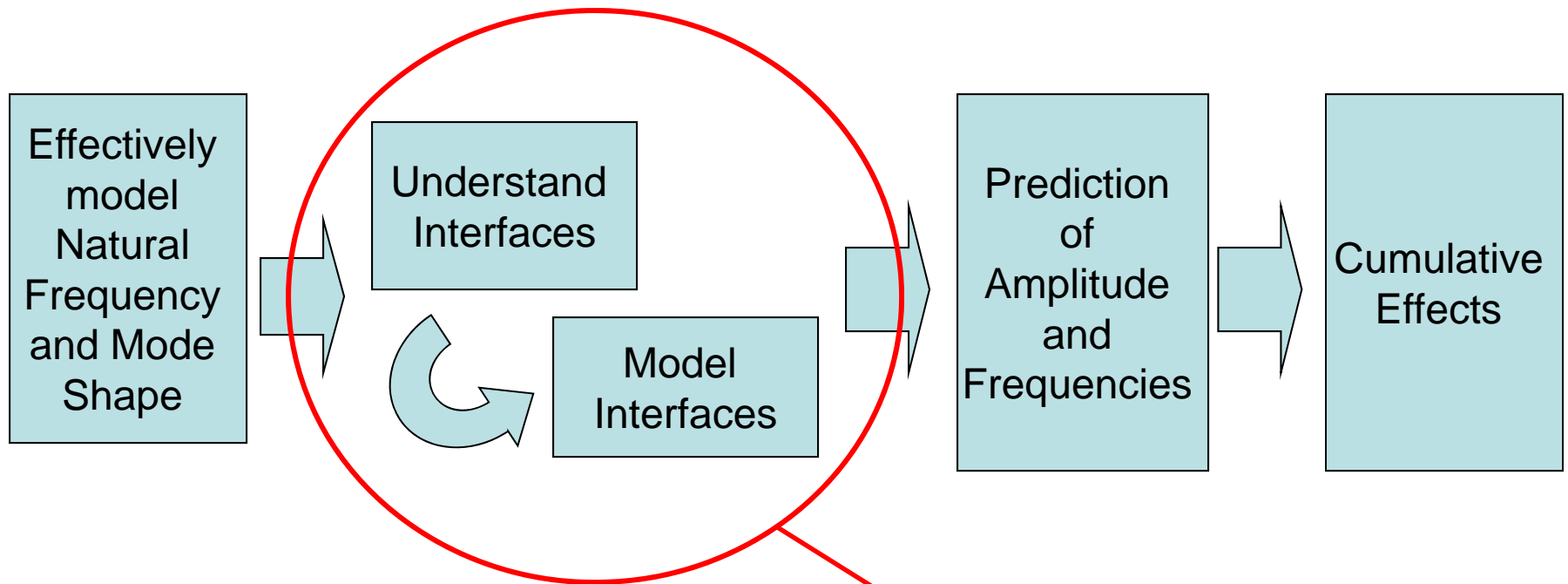
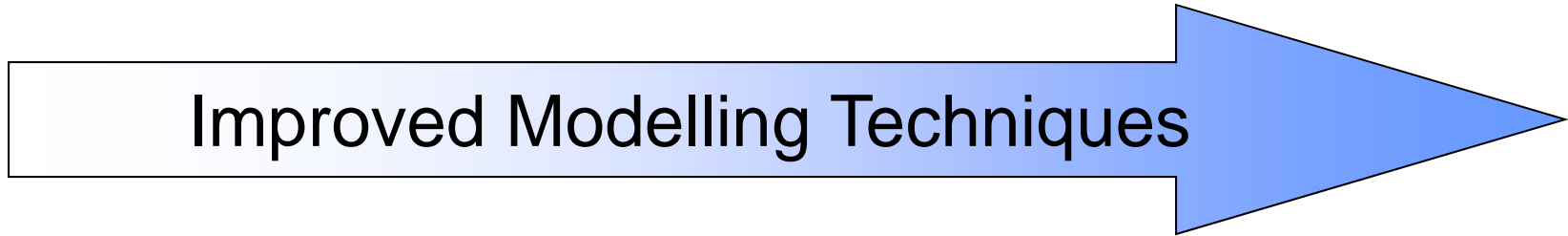
**We can model individual joints (crudely)
and insert them into a system model**

— Experiment
— Model





What Next for Such Interfaces?



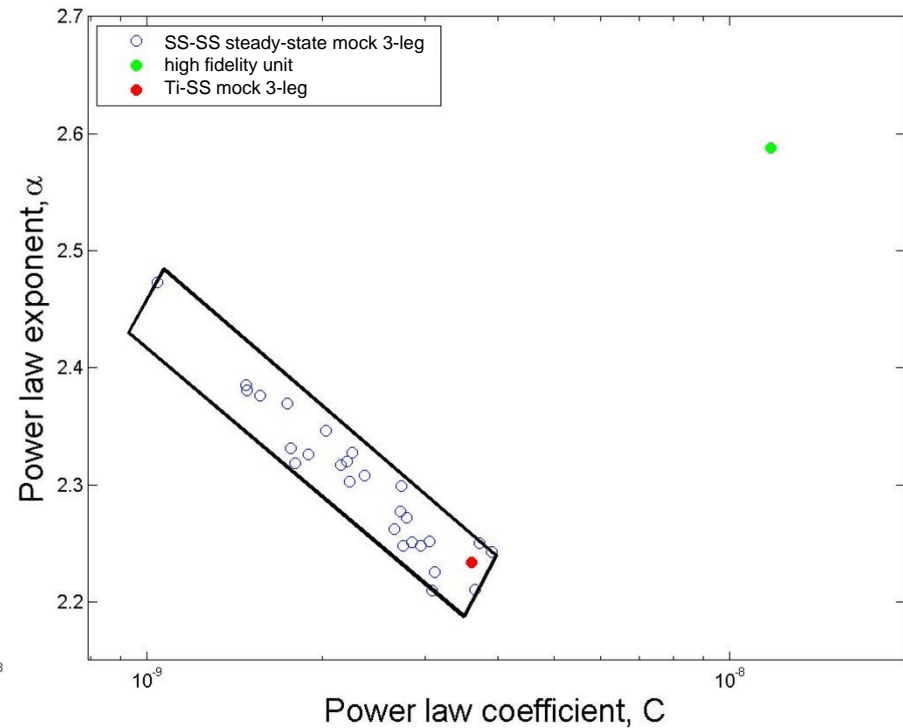
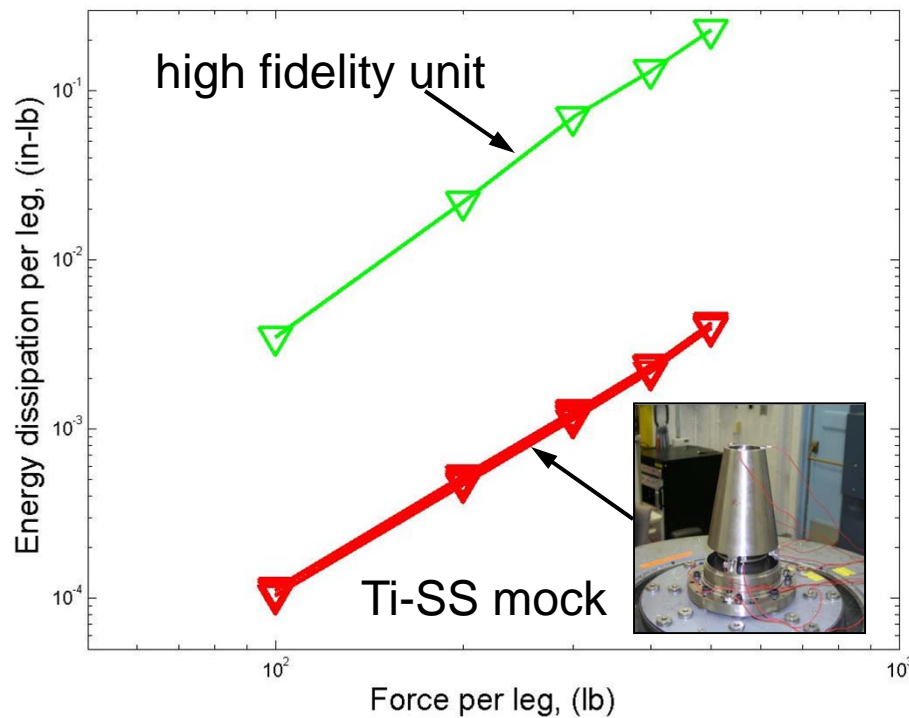
Easier Said Than Done!



The Problem is Larger than Just
an Occasional Lap Joint



Even Whole Subsystems May Behave in Joint-Like Manner



- The dissipation of the high-fidelity unit is very joint-like in nature.
- That dissipation is much more than can be explained by the forward mount joints alone.



Weapons systems contain a plethora of interfaces; How can we account for them in aggregate?

$$M\ddot{u} + C\dot{u} + Ku = F_X(t) + F_J(t, \{x_k^j\})$$

where F_J is force vector for joints and $\{x_k^j\}$ are state variables for joint j

$$\text{Postulate } F_J = M\Phi \left\{ \mathcal{G}_j(\alpha_j(\tau), \tau = -\infty, t) \right\}$$

where α_j are modal coordinates

$$\text{For modal BP11, } \mathcal{G} = \int_0^{\infty} \text{diag}(\{\rho_k(\phi)\}) \beta(t, \phi) d\phi$$

where

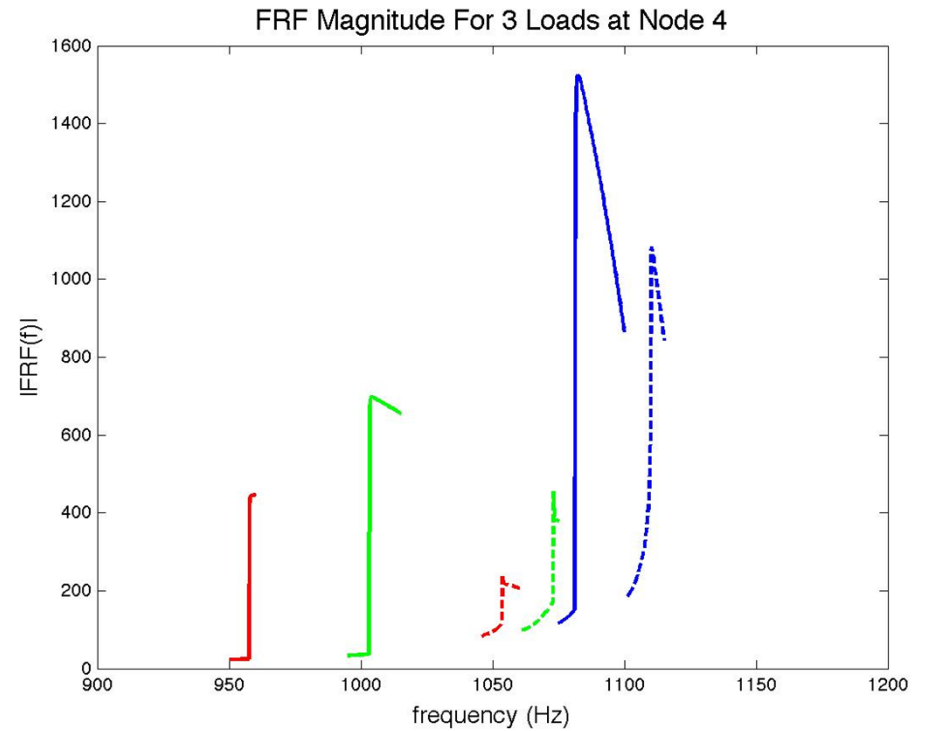
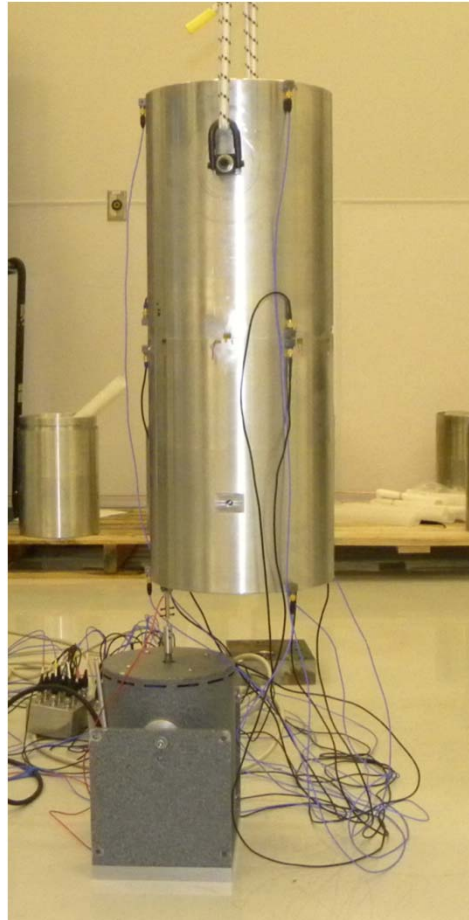
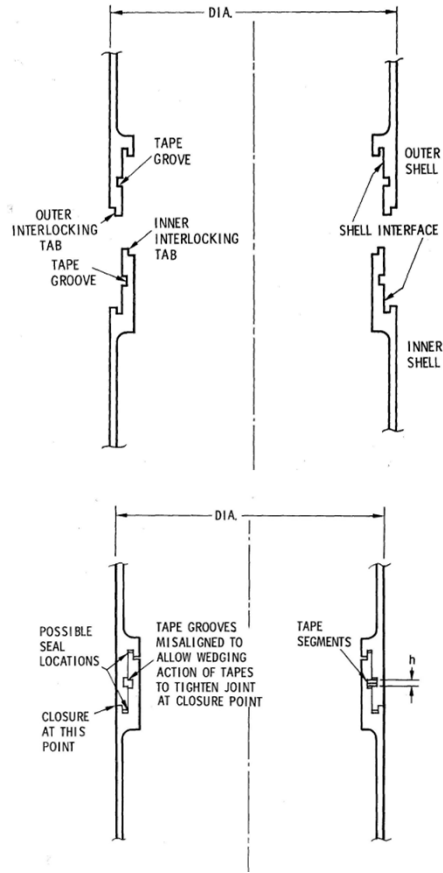
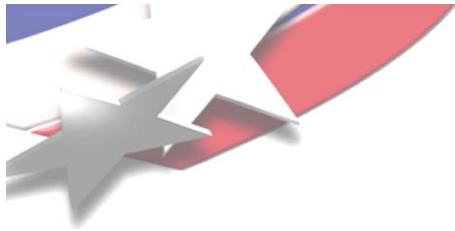
$$\dot{\beta}_k(t, \phi) = \begin{cases} \dot{\alpha} & \text{where } \dot{\alpha}(\alpha_k - \beta_k) > 0 \text{ and } |\alpha_k - \beta_k| = \phi \\ 0 & \text{otherwise} \end{cases}$$



How could we possibly determine the parameters for our nonlinear modal operators?

- Decompose the response in modal components
Look to empirical mode decomposition.
- Fit modal parameters in same way that joint parameters were fit.

Other Sorts of Nonlinear Joint: Consider Tape Joints

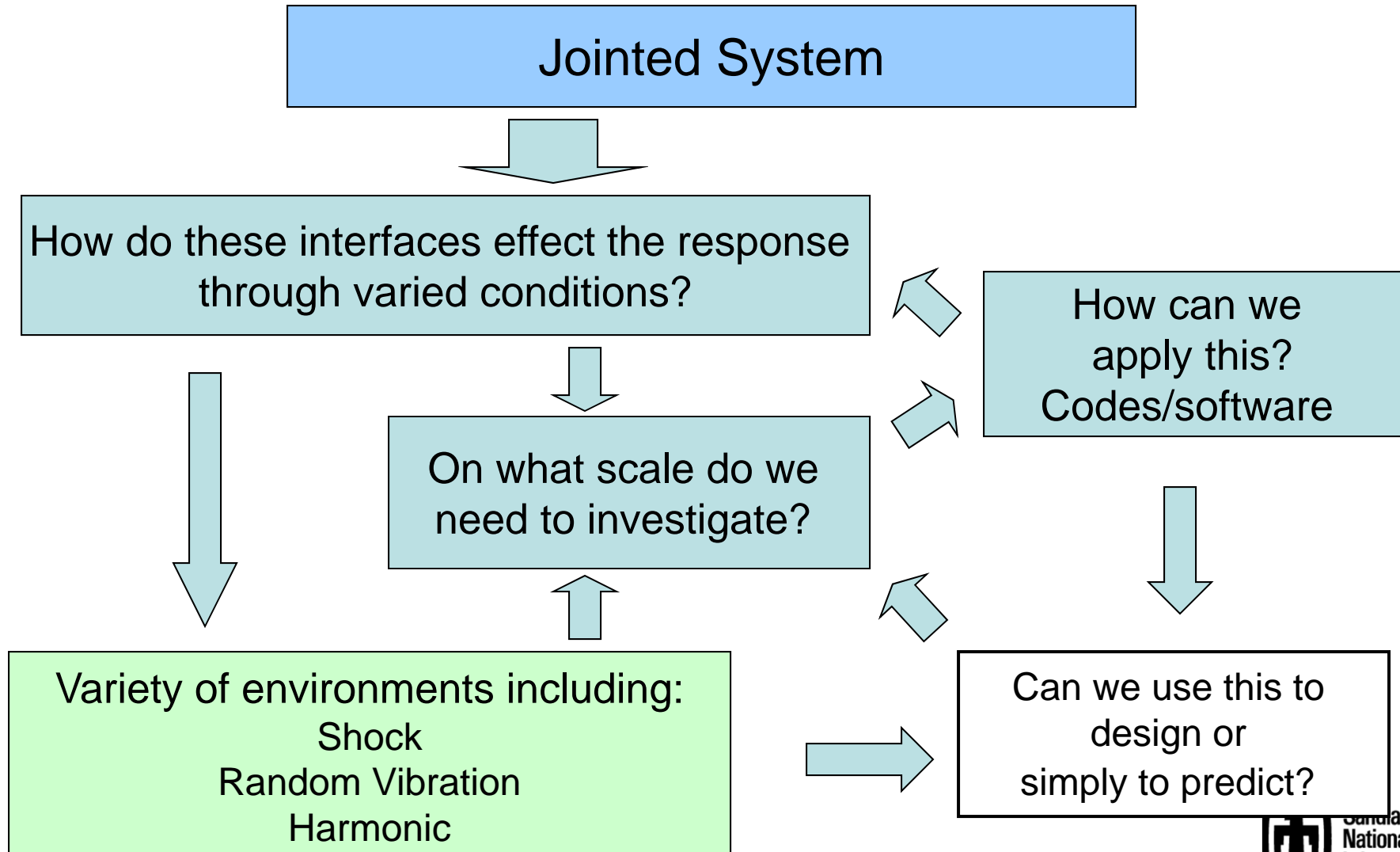


- Multiple FRF show system is very nonlinear
- Shows classic features of softening system

Response is more like that of a Duffing oscillator than that of a linear system



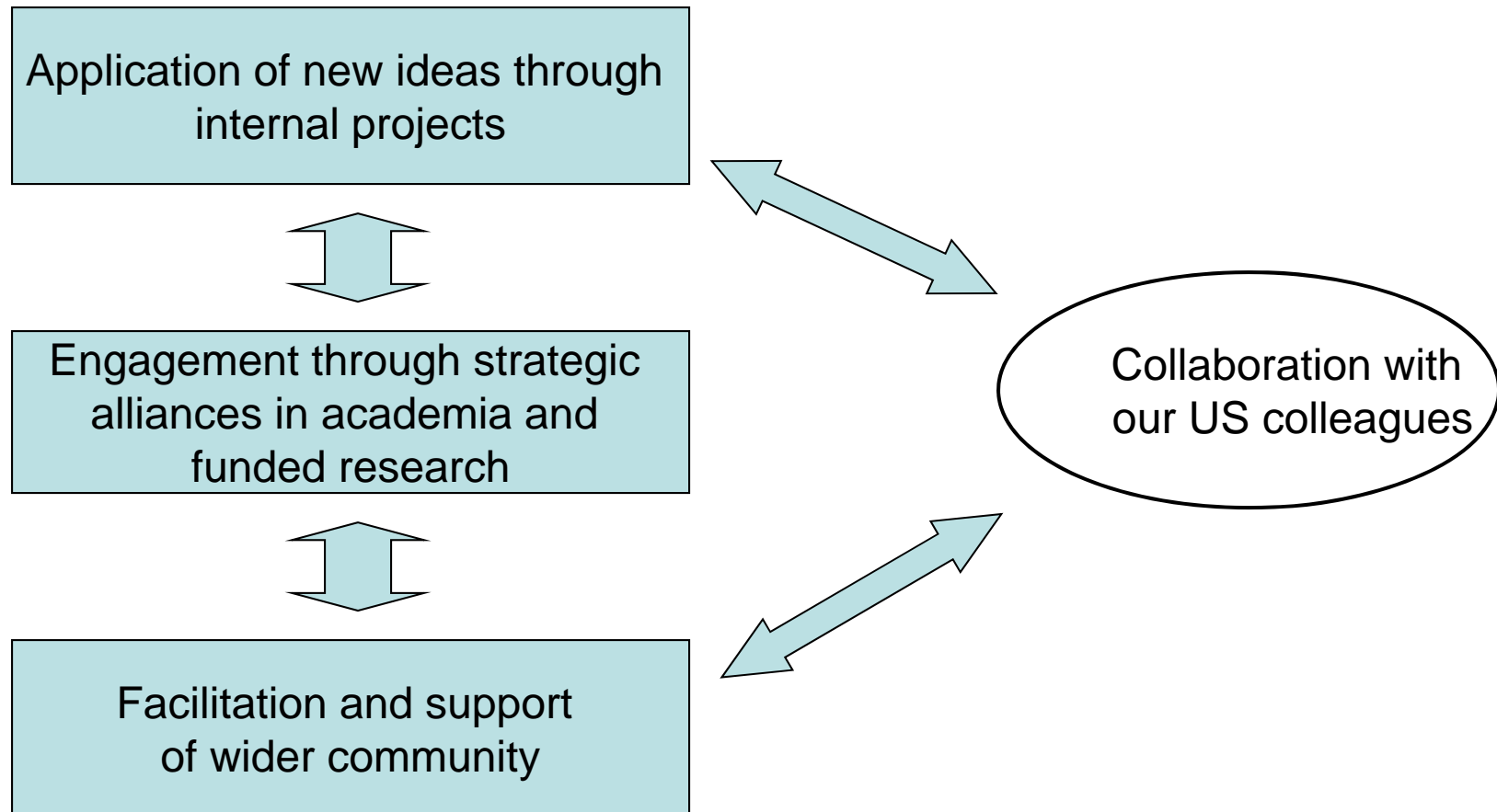
Assessing Where We Stand





How to Move Forward?

- We do not have the resources to commit to significant and sustained in house research...





BACKUP